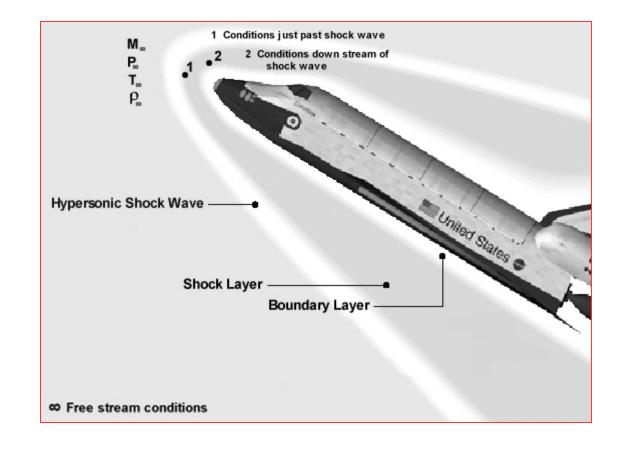


Mechendeel & Ferospece Engineering

Appendix to Section 8: Space Shuttle Tile Thermal Protection System





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Temperature Versus Heat (1)

• Often the concepts of heat and temperature are thought to be the same, but they are not.

• *Temperature* is a number that is related to the average kinetic energy due to the random motion of each molecule of a substance.

In Kelvin degrees, T is directly proportional to the average kinetic energy of the molecules.

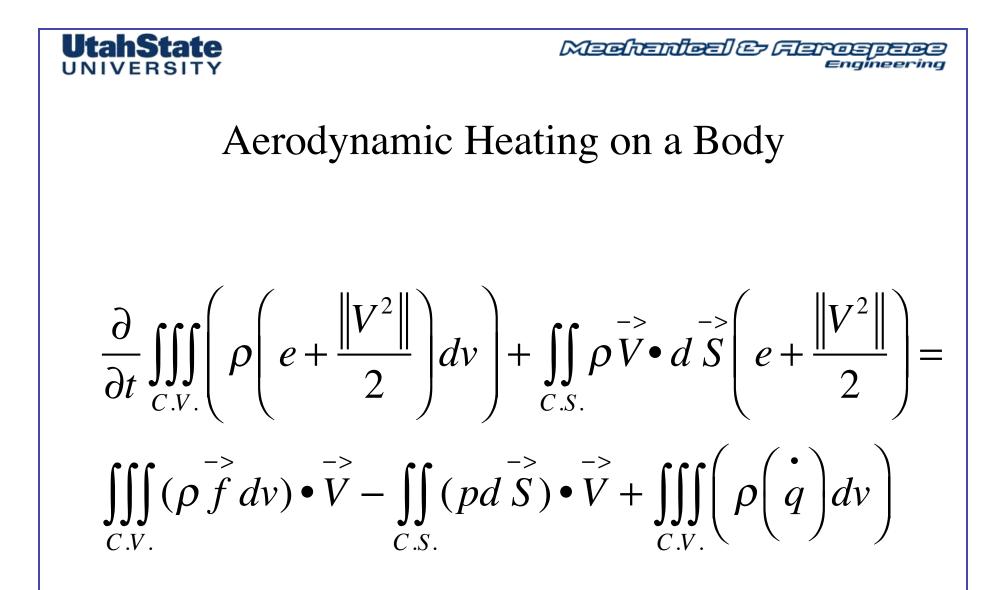
• *Heat* is a measurement of the total energy in a substance. That total energy is made up of not only of the *kinetic energies* of the molecules of the substance, but total energy is also made up of the *potential energies* of the molecules. **MAE 5420** *Compressible Fluids*

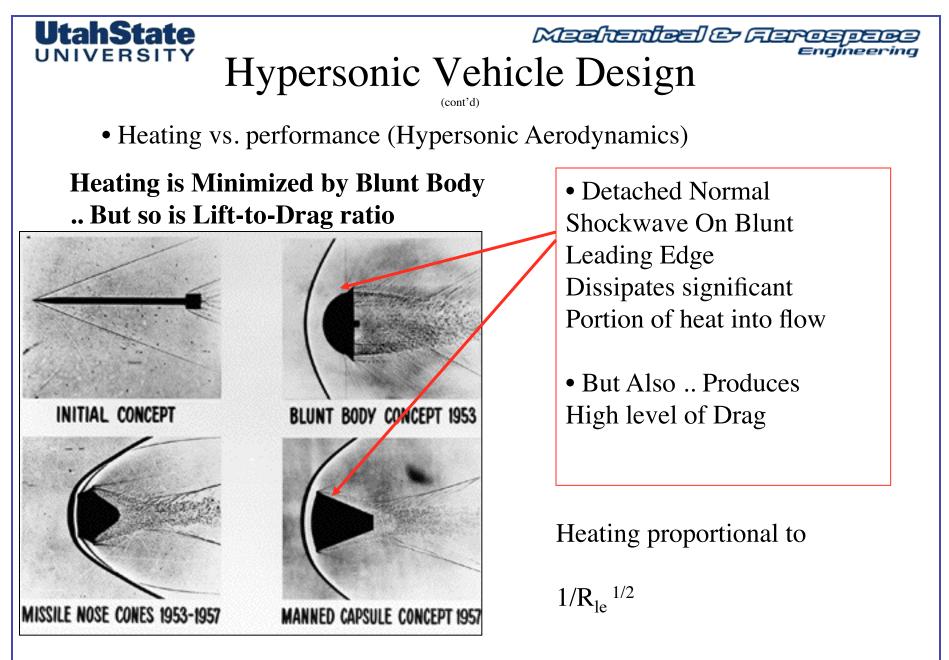


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Temperature Versus Heat (2)

- When heat, (i. e., energy), goes into a substance one of two things can happen:
- 1. The substance can experience a raise in temperature. That is, the heat can be used to speed up the molecules of the substance.
- 2. The substance can change state. For example, if the substance is ice, it can melt into water. This change does not cause a raise in temperature. The energy is used to change the bonding between the molecules. Ablative heat shields use this principal to protect reentering spacecraft
- 3. The degree of temperature change for a given heat input (loss) is the *Heat Capacity*



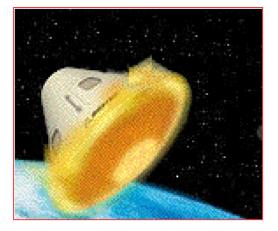




Stagnation Temperature for the Adiabatic Flow of a Calorically Perfect Gas (cont'd)

- Stagnation temperature is a measure of the Kinetic Energy of the flow Field.
- Largely responsible for the high Level of heating that occurs on high speed aircraft or reentering space Vehicles ...

$$T_0 = T \cdot \left(1 + \frac{\gamma - 1}{2} \cdot M^2\right)$$



MAE 5420 - Compressible Fluid Flow

UtahStat

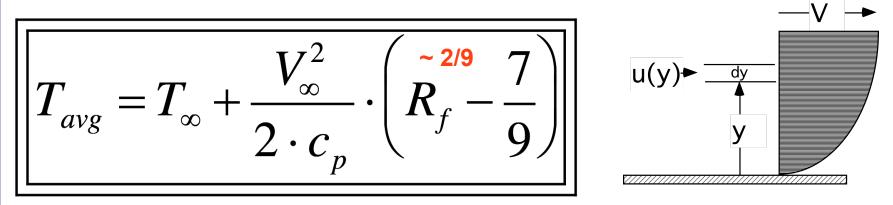
UtahState Mechanical & Flavograd **Compressibility Effects** UNIVERSITY on Skin Friction Model (11)

• Evaluating the Integral

$$T_{avg} = T_{\infty} + R_f \frac{V_{\infty}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left[\left(\frac{y}{\delta} \right)^{1/7} \right]^2 \right) \right] \cdot dy = T_{\infty} + R_f \frac{V_{\infty}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} = T_{\alpha} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} = T_{\alpha} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right) \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot dy = T_{\alpha vg} + R_f \frac{V_{\alpha}^2}{2 \cdot c_p} \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \int_0^{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_f} \left(\frac{y}{\delta} \right)^{2/7} \right] \right] \cdot \frac{1}{\delta} \left[\left(1 - \frac{1}{R_$$

Engineering

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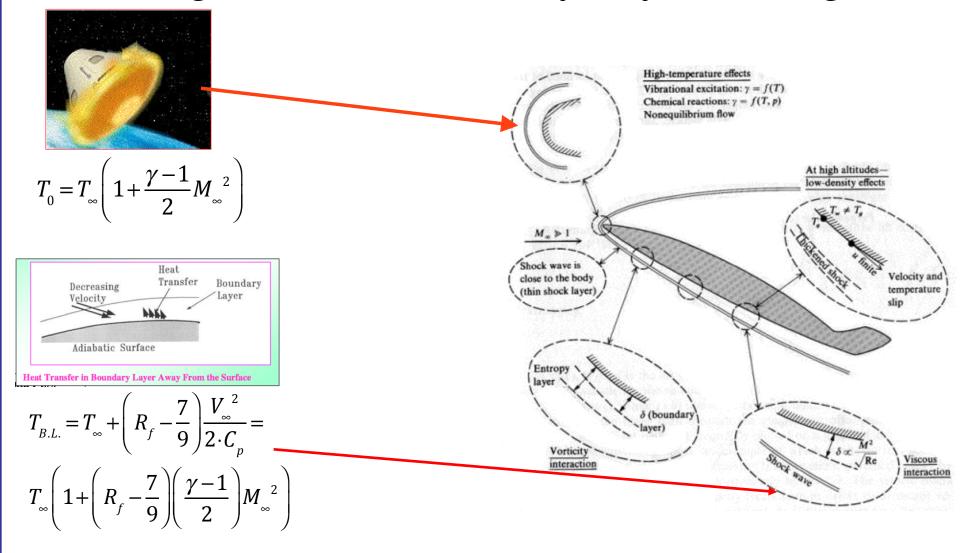
• Valid for 1/7 power Turbulent Flow

MAE 5420 - Compressible Fluid Flow

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Stagnation vs. Boundary Layer heating





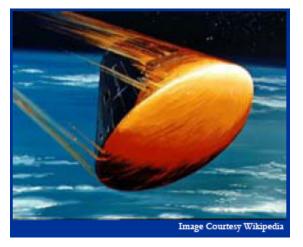
Mechanical & Ferospece Ablation Example

-Heat shield consisting of phenolic resin in a metal "honeycomb" At high heat flux, resin

-Material decomposes via pyrolysis absorbing heat

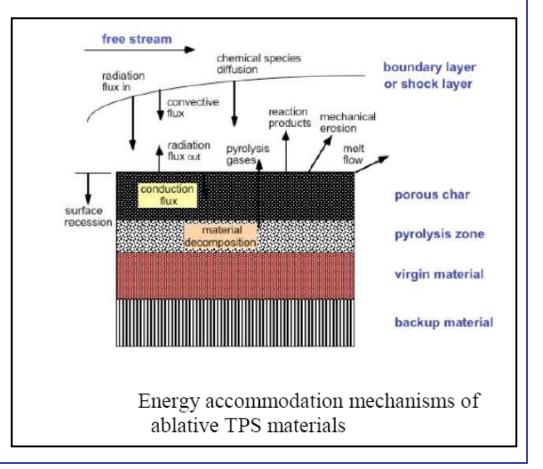
-Products form a barrier between hot gasses and spacecraft structure

-Surface temperature remains low



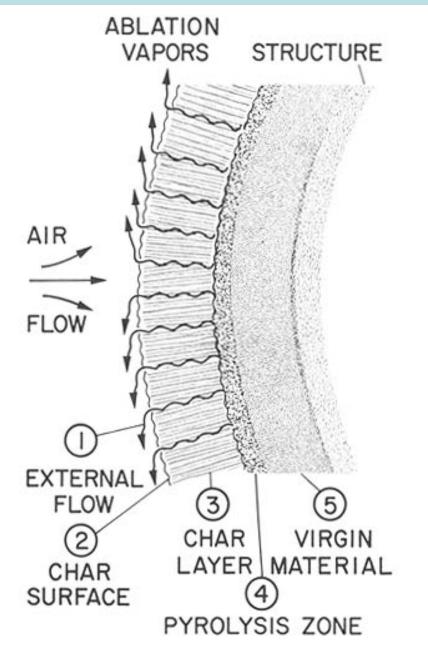
Laub, B. Thermal Protection Technology and Facility Needs for Demanding Future Planetary Missions, NASA Ames Research Center, October 2003

MAE 5420 Compressible Fluids



Engineer

Heat blockage by mass transfer



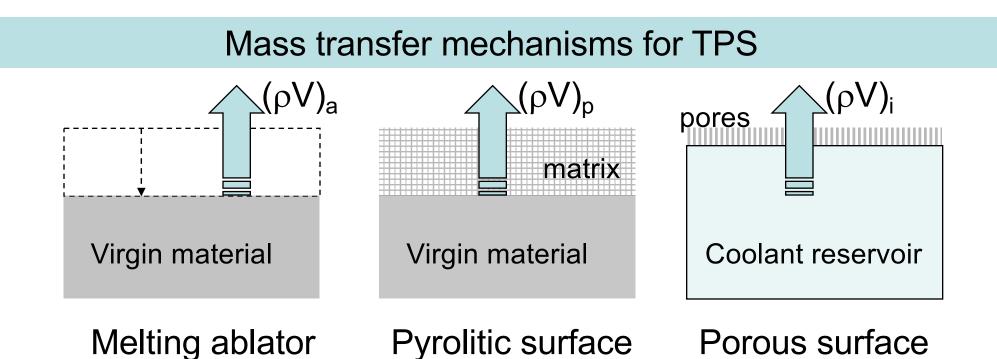
•Direct injection of a coolant through a porous surface

•Vaporization of melted surface material

•Sublimation of the solid surface material

•Pyrolysis of surface material

Space Access Vehicle Design

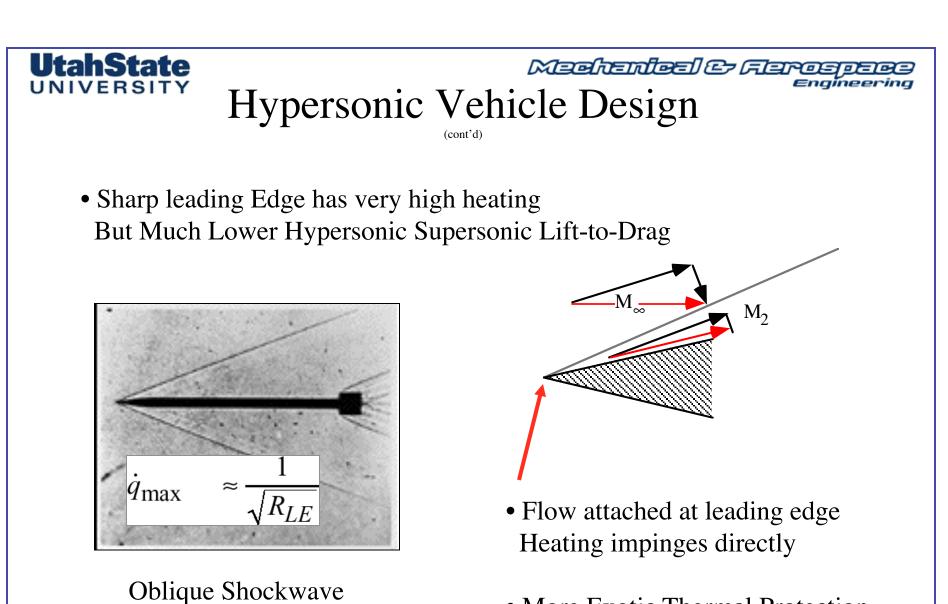


$$(\rho V)_a Q_a^* = (\rho V)_p Q_p^* = (\rho V)_i (h_i - h_y) = q_{c,w}$$

For Teflon, (in Btu/lb) an effective heat of ablation is

$$Q^* \approx 0.48 \left(h_{s,e} - h_w \right) + 564$$

Space Access Vehicle Design



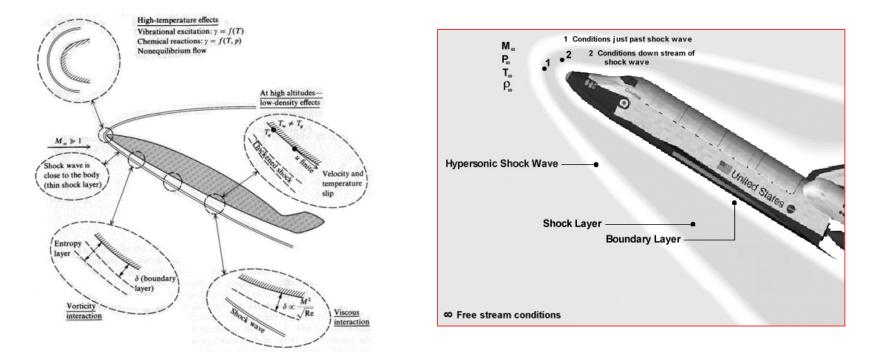
• More Exotic Thermal Protection Systems Required

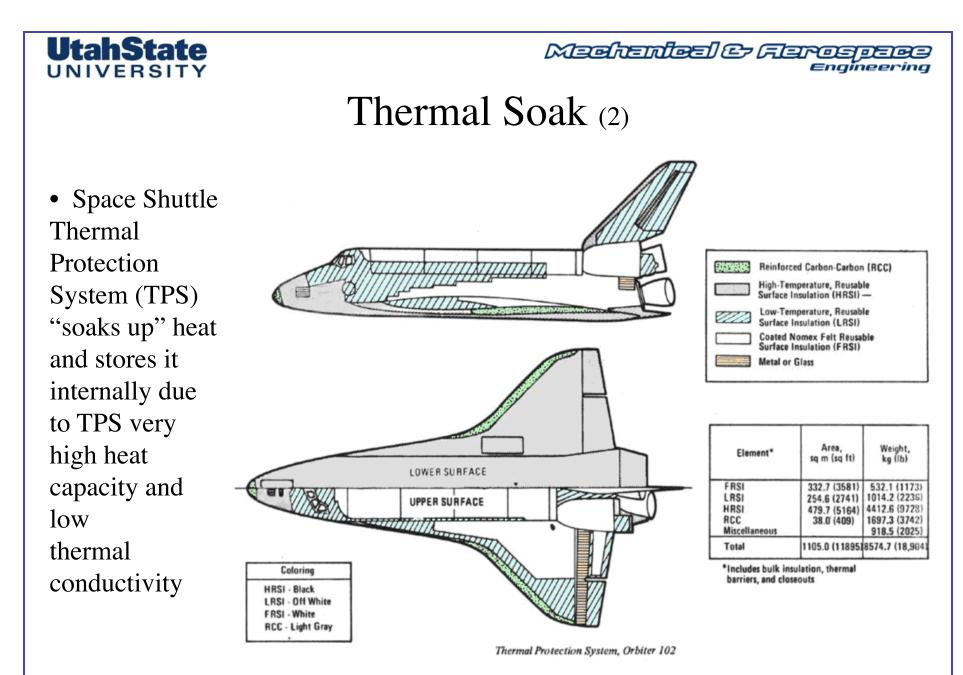


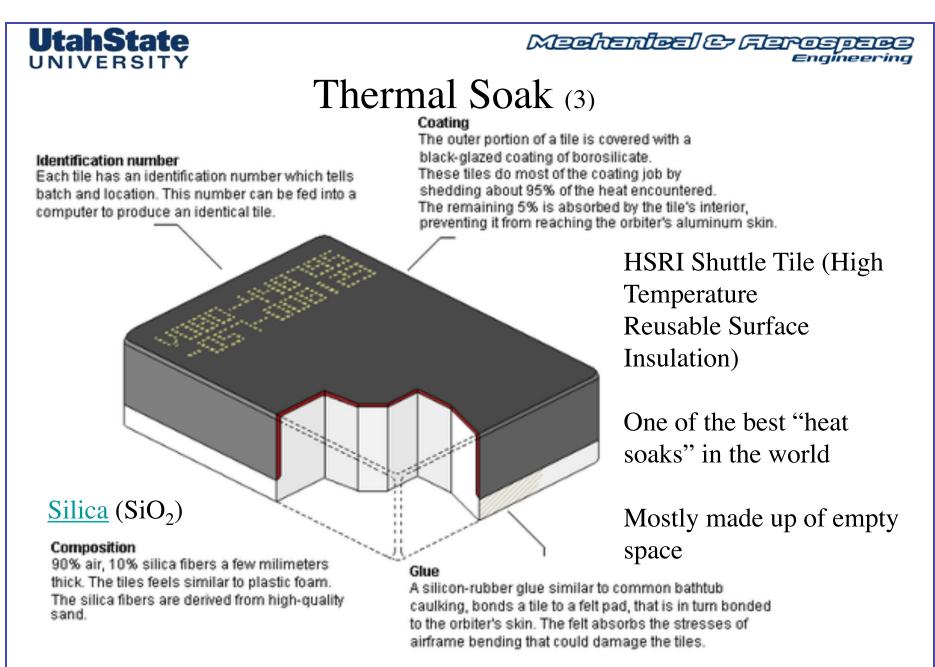
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Thermal Soak

• Space Shuttle Thermal Protection System "soaks up" heat and stores it internally due to its very high heat capacity and low thermal conductivity







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Thermal Soak (4)



Density	144.2 kg/m ³ (9 lb/ft ³ LI-900) 352.5 kg/m ³ (22 lb/ft ³ LI-2200)	
Specific heat).628 KJ/kg-K (0.15 BTU/lb-°F)	
Thermal conductivity	0.0485 W/m-k (0.028 BTU/ft-hr-°F) at 21 °C)	
	0.126 W/m-k (0.073 BTU/ft-hr-°F at 1093 °C)	

Maximum reuse temperature

>1260 °C

Maximum single 1538 °C use temperature

Reusability at 2300 °F

>100 missions

MAE 5420 Compressible Fluids

Material	Thermal conductivity
	<u>W/(m·K</u>)
Shuttle Tile (LI-900)	0.048-0.126
Air	0.025
Rubber	0.16
Thermal grease	0.7 - 3
Thermal <u>epoxy</u>	1 - 7
<u>Glass</u>	1.1
Concrete, stone	1.7
Sandstone	2.4
Stainless steel	$12.11 \sim 45.0$
Lead	35.3
<u>Aluminium</u>	220 (pure) 120180 (alloys)
Gold	318
Copper	380
<u>Silver</u>	429
Diamond	900 - 2320

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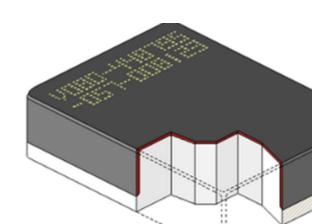
More or Less ... Only Air is a better insulator (except for exotic materials like aero gels)

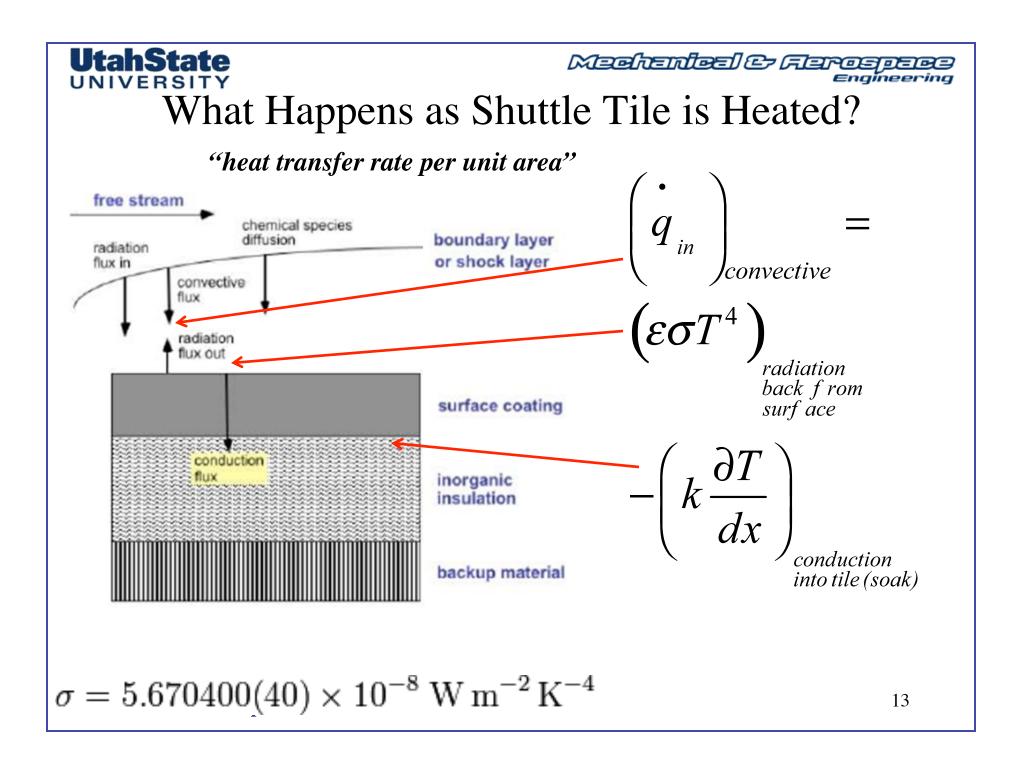
... a copper penny conducts heat almost 7000 Times faster than a shuttle tile

Compare Shuttle Tile Thermal Conductivity to Conventional Materials

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12







Because thermal conductivity of shuttle tile is so low ... heat is radiated back from the surface faster than it is absorbed into the body

- --- Assume 1260 C surface temperature
- --- 80 C interior wall temperature
- --- 10 cm thick tile

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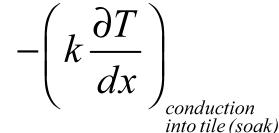
Always work in absolute temperature units

Mechanical & Ferosoa

$$\left(\mathcal{E}\sigma T^{4}\right)_{radiation} = 0.85 \cdot 5.6704 \times 10^{-8} \left(1260 + 273\right)^{4} \left(\frac{1}{100}\right)^{2} = 26.62 \text{ W/cm}^{2}$$

radiation back f rom surf ace

Tile radiates back 180 times more heat than it Conducts into the structure!



$$= 0.126 \frac{(1260 - 80)}{0.1} \left(\frac{1}{100}\right)^2 = 0.149 \text{ W/cm}^2$$

"heat transfer rate per unit area"

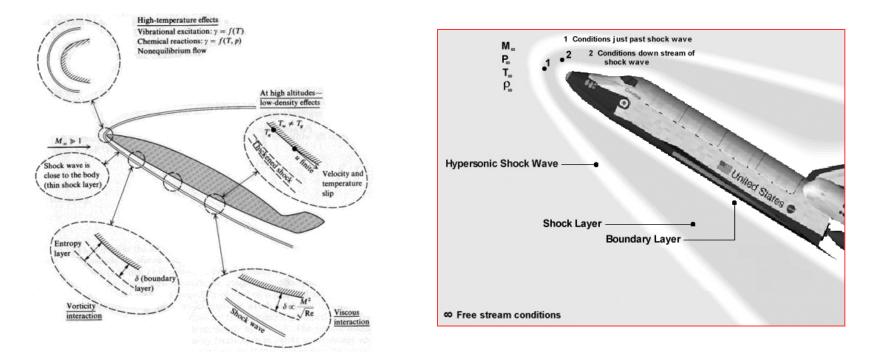
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Thermal Soak (Revisited)

• Space Shuttle Thermal Protection System "soaks up" heat and stores it internally due to its very high heat capacity and low thermal conductivity





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What Happens as Shuttle Tile is heated? (3)

But the Shuttle Tiles Still Stored a Lot of Heat That Had to be Removed Post Landing!



UtahState UNIVERSITY What Happens when a penny is heated? (2) • Wall gradient is rapidly neutralized and whole penny heats up

... no insulation properties at all

